

AN ANALYTICAL AND NUMERICAL APPROACH TO A BILATERAL CONTACT PROBLEM WITH NONMONOTONE FRICTION

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We consider a mathematical model which describes the contact between a linearly elastic body and an obstacle, the so-called foundation. The process is static and the contact is bilateral, i.e., there is no loss of contact. The friction is modeled with a nonmonotone law. The purpose of this work is to provide an error estimate for the Galerkin method as well as to present and compare two numerical methods for solving the resulting nonsmooth and nonconvex frictional contact problem. The first approach is based on the nonconvex proximal bundle method, whereas the second one deals with the approximation of a nonconvex problem by a sequence of nonsmooth convex programming problems. Some numerical experiments are realized to compare the two numerical approaches.

Keywords: linearly elastic material, bilateral contact, nonmonotone friction law, hemivariational inequality, finite element method, error estimate, nonconvex proximal bundle method, quasi-augmented Lagrangian method, Newton method.

1. Introduction

This paper deals with numerical solution for frictional contact problems governed by nonmonotone friction laws. Two independent approaches inspired by the finite element method are applied and numerical results are presented for a simple two-dimensional model problem. The first approach is based on minimization of the energy functional by means of the Proximal Bundle Method (PBM) (see the works of Mäkelä (1990; 2001) for a survey of bundle methods for nonsmooth optimization), while the second one relies on approximation of a nonconvex problem by a sequence of convex ones. The main theoretical result of the paper is Theorem 3, which provides the error estimate between the Galerkin solution and the exact one. This result is stronger than those previously obtained in the literature. It provides the strong convergence of subsequence of Galerkin solutions (see Haslinger *et al.*, 1999, Theorem 3.12) and it is a generalization of the classical Cea lemma. The estimate is possible due to the one sided Lipschitz condition imposed on the friction bound, which is the natural assumption in frictional problems. Analogous results for the convex case

are available in the works of Hild and Renard (2007) as well as Han and Sofonea (2002).

In one approach to numerical approximation of the solution based on the PBM, the Galerkin problem is converted into the form of minimization of a not necessarily convex energy functional. In each iteration step a piecewise linear approximation of the objective functional is constructed and it is regularized by adding a quadratic term. Bundle methods have already been used to solve the elastic contact problem with friction and a normal response given by a nonmonotone law by Miettinen (1995), while the delamination problem for laminated composite with nonmonotone adhesive force between laminae is solved with bundle methods by Mäkelä *et al.* (1999). The review of these results can be found in the monograph of Haslinger *et al.* (1999).

Another approach to the numerical solution of a nonconvex frictional problem consists in solving a sequence of auxiliary convex problems which approximate the original one. Each convex problem is solved by a numerical strategy based on a quasi-Lagrangian formulation combined with a Newton

method. The approach based on the sequence of convex problems was used to solve the contact problem with nonmonotone softening behavior of the binding material in the normal direction by Tzaferopoulos et al. (1995), while the elastic contact problem with nonmonotone friction was solved by Mistakidis and Panagiotopoulos (1997; 1998). To the best of our knowledge, the literature concerning the numerical solution of nonmonotone friction problems is very limited. The bundle Newton approach was used for unilateral problems by Baniotopoulos et al. (2005).

The paper is organized as follows. In Section 2 we introduce some notations and preliminaries. In Section 3 we present the frictional contact problem together with its variational formulation. Existence and uniqueness results are also briefly presented. Section 4 is devoted to the presentation of the Galerkin problem which is the basis for the numerical schemes used to approximate the solution. The theorem on the error estimate is provided. In Section 5 we present the nonconvex PBM for the solution of the bilateral contact problem with nonmonotone friction. Section 6 concerns the solution of the nonsmooth and nonconvex problem by using a numerical strategy based on a sequence of convex programming problems. In the last section we present some numerical examples to compare the two preceding numerical methods for the solution of bilateral contact problems with a nonmonotone friction law.

2. Notation and preliminaries

In this section we present the notation we shall use along with some preliminary material. For further details we refer the reader to the works of Duvaut and Lions (1976), Ionescu and Sofonea (1993), as well as Panagiotopoulos (1985).

We denote by \mathbb{S}^d the space of second order symmetric tensors on \mathbb{R}^d ($d \leq 3$ in applications), while “ \cdot ” and “ $|\cdot|$ ” will represent the inner product and the Euclidean norm on \mathbb{S}^d and \mathbb{R}^d , respectively, i.e.,

$$\begin{aligned} \mathbf{u} \cdot \mathbf{v} &= u_i v_i, & |\mathbf{v}| &= (\mathbf{v} \cdot \mathbf{v})^{\frac{1}{2}}, \quad \forall \mathbf{u}, \mathbf{v} \in \mathbb{R}^d, \\ \boldsymbol{\sigma} \cdot \boldsymbol{\varepsilon} &= \sigma_{ij} \varepsilon_{ij}, & |\boldsymbol{\varepsilon}| &= (\boldsymbol{\varepsilon} \cdot \boldsymbol{\varepsilon})^{\frac{1}{2}}, \quad \forall \boldsymbol{\sigma}, \boldsymbol{\varepsilon} \in \mathbb{S}^d. \end{aligned}$$

Here and below the indices i and j run between 1 and d , and the summation convention over repeated indices is adopted.

Let $\Omega \subset \mathbb{R}^d$ be a bounded domain with a Lipschitz boundary Γ , and let \mathbf{n} denote the unit outer normal on Γ and $\boldsymbol{\tau}$ the associated tangent vector. We shall use the notation

$$H = L^2(\Omega)^d = \{ \mathbf{u} = (u_i) \mid u_i \in L^2(\Omega) \},$$

$$\begin{aligned} Q &= \{ \boldsymbol{\sigma} = (\sigma_{ij}) \mid \sigma_{ij} = \sigma_{ji} \in L^2(\Omega) \}, \\ H_1 &= \{ \mathbf{u} = (u_i) \mid \boldsymbol{\varepsilon}(\mathbf{u}) \in Q \}, \\ Q_1 &= \{ \boldsymbol{\sigma} \in Q \mid \text{Div } \boldsymbol{\sigma} \in H \}. \end{aligned}$$

Here $\boldsymbol{\varepsilon} : H_1 \rightarrow Q$ and $\text{Div} : Q_1 \rightarrow H$ are the deformation and divergence operators, respectively, defined by

$$\begin{aligned} \boldsymbol{\varepsilon}(\mathbf{u}) &= (\varepsilon_{ij}(\mathbf{u})), \quad \varepsilon_{ij}(\mathbf{u}) = \frac{1}{2}(u_{i,j} + u_{j,i}), \\ \text{Div } \boldsymbol{\sigma} &= (\sigma_{ij,j}), \end{aligned}$$

where the index that follows a comma indicates a partial derivative with respect to the corresponding component of the independent variable. The spaces H , Q , H_1 and Q_1 are real Hilbert spaces endowed with the canonical inner products given by

$$\begin{aligned} (\mathbf{u}, \mathbf{v})_H &= \int_{\Omega} u_i v_i \, dx, \\ (\boldsymbol{\sigma}, \boldsymbol{\varepsilon})_Q &= \int_{\Omega} \sigma_{ij} \varepsilon_{ij} \, dx, \\ (\mathbf{u}, \mathbf{v})_{H_1} &= (\mathbf{u}, \mathbf{v})_H + (\boldsymbol{\varepsilon}(\mathbf{u}), \boldsymbol{\varepsilon}(\mathbf{v}))_Q, \\ (\boldsymbol{\sigma}, \boldsymbol{\varepsilon})_{Q_1} &= (\boldsymbol{\sigma}, \boldsymbol{\varepsilon})_Q + (\text{Div } \boldsymbol{\sigma}, \text{Div } \boldsymbol{\varepsilon})_H. \end{aligned}$$

The associated norms on these spaces are further denoted by $\|\cdot\|_H$, $\|\cdot\|_Q$, $\|\cdot\|_{H_1}$ and $\|\cdot\|_{Q_1}$, respectively.

Let $H_{\Gamma} = H^{1/2}(\Gamma)^d$ and let $\gamma : H_1 \rightarrow H_{\Gamma}$ be the trace map. For every element $\mathbf{v} \in H_1$ we still write \mathbf{v} to denote the trace $\gamma\mathbf{v}$ of \mathbf{v} on Γ , and we denote by v_n and v_{τ} the normal and tangential components of \mathbf{v} on the boundary Γ given by

$$v_n = \mathbf{v} \cdot \mathbf{n}, \quad v_{\tau} = \mathbf{v} - v_n \mathbf{n}. \tag{1}$$

Let H_{Γ}^* be the dual of H_{Γ} and let (\cdot, \cdot) denote the duality pairing between H_{Γ}^* and H_{Γ} . For every $\boldsymbol{\sigma} \in Q_1$ there exists an element $\boldsymbol{\sigma}\mathbf{n} \in H_{\Gamma}^*$ such that

$$(\boldsymbol{\sigma}, \boldsymbol{\varepsilon}(\mathbf{v}))_Q + (\text{Div } \boldsymbol{\sigma}, \mathbf{v})_H = (\boldsymbol{\sigma}\mathbf{n}, \gamma\mathbf{v}), \quad \forall \mathbf{v} \in H_1. \tag{2}$$

Moreover, if $\boldsymbol{\sigma}$ is a smooth (say C^1) function, then

$$(\boldsymbol{\sigma}\mathbf{n}, \gamma\mathbf{v}) = \int_{\Gamma} \boldsymbol{\sigma}\mathbf{n} \cdot \mathbf{v} \, d\Gamma, \quad \forall \mathbf{v} \in H_1. \tag{3}$$

We also denote by σ_n and σ_{τ} the normal and tangential traces of $\boldsymbol{\sigma}$ and we recall that, when $\boldsymbol{\sigma}$ is smooth enough, then

$$\sigma_n = (\boldsymbol{\sigma}\mathbf{n}) \cdot \mathbf{n}, \quad \sigma_{\tau} = \boldsymbol{\sigma}\mathbf{n} - \sigma_n \mathbf{n}. \tag{4}$$

Finally, we recall definitions of the generalized derivative and gradient (see Clarke, 1983). Let X be a reflexive Banach space and X^* its dual. The Clarke generalized directional derivative of a locally Lipschitz

function $h : X \rightarrow \mathbb{R}$ at the point $x \in X$ in the direction $v \in X$, denoted by $h^0(x; v)$, is defined by

$$h^0(x; v) = \limsup_{y \rightarrow x, \lambda \downarrow 0} \frac{h(y + \lambda v) - h(y)}{\lambda}.$$

The Clarke subdifferential of h at x denoted by $\partial h(x)$ is a subset of X^* given by $\partial h(x) = \{\zeta \in X^* : h^0(x; v) \geq \langle \zeta, v \rangle_{X^* \times X} \text{ for all } v \in X\}$.

3. Mechanical problem and variational formulation

In this section we describe the model for the nonmonotone frictional bilateral contact, present the variational formulation well suited for the numerical treatment used in the sequel, and finally recall an existence and uniqueness result.

The physical setting is as follows. A linearly elastic body occupies an open bounded connected set $\Omega \subset \mathbb{R}^d$ ($d \leq 3$ in applications) with a Lipschitz boundary Γ that is partitioned into three disjoint parts $\bar{\Gamma}_1, \bar{\Gamma}_2$ and $\bar{\Gamma}_3$ with Γ_1, Γ_2 and Γ_3 being relatively open, and $\text{meas}(\Gamma_1) > 0$. The body is clamped on Γ_1 and thus the displacement field vanishes there. A volume force of density \mathbf{f}_0 acts in Ω and a surface traction of density \mathbf{f}_2 acts on Γ_2 . The body is in frictional contact with an obstacle on Γ_3 . We assume that there is no loss of contact during the process, i.e., the contact is bilateral. Thus, the normal displacement u_n vanishes on Γ_3 . We model the friction by a nonmonotone friction law. The material is linearly elastic and the process is assumed to be static.

In the study of the frictional contact problem we need the following assumptions on its data:

- $H(\mathcal{E})$: the elasticity operator $\mathcal{E} : \Omega \times \mathbb{S}^d \rightarrow \mathbb{S}^d$ is a bounded symmetric positive definite fourth order tensor, i.e.,

$$\left\{ \begin{array}{l} \text{(a) } \mathcal{E}_{ijkl} \in L^\infty(\Omega), \quad 1 \leq i, j, k, l \leq d; \\ \text{(b) } \mathcal{E}\boldsymbol{\sigma} \cdot \boldsymbol{\tau} = \boldsymbol{\sigma} \cdot \mathcal{E}\boldsymbol{\tau}, \quad \forall \boldsymbol{\sigma}, \boldsymbol{\tau} \in \mathbb{S}^d, \text{ a.e. in } \Omega; \\ \text{(c) } \mathcal{E}\boldsymbol{\tau} \cdot \boldsymbol{\tau} \geq m|\boldsymbol{\tau}|^2, \quad \forall \boldsymbol{\tau} \in \mathbb{S}^d, \text{ a.e. in } \Omega \\ \text{with } m > 0. \end{array} \right. \quad (5)$$

- $H(f)$: the force and the traction densities satisfy

$$\mathbf{f}_0 \in L^2(\Omega)^d, \quad \mathbf{f}_2 \in L^2(\Gamma_2)^d. \quad (6)$$

- $H(\mu)$: the friction bound satisfies

$$\left\{ \begin{array}{l} \text{(a) } \mu : [0, \infty) \rightarrow \mathbb{R} \text{ is measurable;} \\ \text{(b) } \mu(0) = \lim_{x \rightarrow 0^+} \mu(x) > 0; \\ \text{(c) } -c_2 \leq \mu(t) \leq c_1(1+t), \quad \forall t \geq 0, \\ \text{with } c_1, c_2 > 0; \\ \text{(d) } \mu(t_1) - \mu(t_2) \geq -\lambda(t_1 - t_2), \quad \forall t_1 > t_2 \geq 0, \\ \text{with } \lambda > 0. \end{array} \right. \quad (7)$$

Remark 1. The hypothesis $H(\mu)$ (c) implies that

$$|\mu(t)| \leq c(1+t), \quad \forall t \geq 0, \quad \text{with } c > 0,$$

and

$$\mu(t) \geq -d(1 + \frac{1}{t}), \quad \forall t > 0, \quad \text{with } d > 0,$$

Remark 2. In a particular case, since μ corresponds to the coefficient of friction, it is nonnegative, so the lower bound in (c) obviously holds. The condition (d) is the so-called one-side Lipschitz condition, which allows the function to decrease with a rate not faster than λ .

The classical formulation of the mechanical problem is the following.

Problem 1. (\mathcal{P}_M) Find a displacement field $\mathbf{u} : \Omega \rightarrow \mathbb{R}^d$ and a stress field $\boldsymbol{\sigma} : \Omega \rightarrow \mathbb{S}^d$ such that

$$\text{Div } \boldsymbol{\sigma} + \mathbf{f}_0 = \mathbf{0} \quad \text{in } \Omega, \quad (8)$$

$$\boldsymbol{\sigma} = \mathcal{E}\boldsymbol{\varepsilon}(\mathbf{u}) \quad \text{in } \Omega, \quad (9)$$

$$\mathbf{u} = \mathbf{0} \quad \text{on } \Gamma_1, \quad (10)$$

$$\boldsymbol{\sigma}\mathbf{n} = \mathbf{f}_2 \quad \text{on } \Gamma_2, \quad (11)$$

$$u_n = 0 \quad \text{on } \Gamma_3, \quad (12)$$

$$\left. \begin{array}{l} |\boldsymbol{\sigma}_\tau| \leq \mu(|\mathbf{u}_\tau|)S \\ -\boldsymbol{\sigma}_\tau = \mu(|\mathbf{u}_\tau|)S \frac{\mathbf{u}_\tau}{|\mathbf{u}_\tau|} \end{array} \right\} \text{ on } \Gamma_3. \quad (13)$$

In (13) $\mu(|\mathbf{u}_\tau|)S$ represents the magnitude of the limiting friction traction at which slip begins. Here, $S \geq 0$ is a given value. The friction bound and, more precisely, the friction coefficient μ depend on the tangential displacement $|\mathbf{u}_\tau|$. The strict inequality in (13) holds in the stick zone and the equality holds in the slip zone. This physical model of slip-dependent friction was introduced by Rabinowicz (1951) for the geophysical context of earthquake modeling, and was also studied by Ionescu and Paumier (1996), Ionescu and Nguyen (2002), Ionescu *et al.* (2003), Shillor *et al.* (2004), as well as Migórski and Ochal (2005). Due to the basic properties of the Clarke subdifferential (cf. Clarke, 1983), the right-hand side of the friction conditions (13) can be written as a subdifferential of a locally Lipschitz,

possibly nonconvex superpotential j_s which depends on the tangential displacement \mathbf{u}_τ . In fact, if the function $j_s : \mathbb{R}^d \rightarrow \mathbb{R}$ is defined by

$$j_s(\boldsymbol{\xi}) = S \int_0^{|\boldsymbol{\xi}|} \mu(t) dt, \quad (14)$$

then we can prove that under assumptions $H(\mu)(a)-(c)$ the conditions (13) are equivalent to the following subdifferential inclusion:

$$-\boldsymbol{\sigma}_\tau \in \partial j_s(\mathbf{u}_\tau) \quad \text{on } \Gamma_3, \quad (15)$$

where $\partial j_s(\boldsymbol{\xi})$ denotes the Clarke subdifferential of j_s . By means of basic calculations, one can easily prove the following lemma on the properties of the function j_s ,

Lemma 1. *If the assumptions $H(\mu)(a)-(c)$ hold, then the function j_s defined by (14) has the following properties:*

- $H(j_s)$:

$$\left\{ \begin{array}{l} (a) \text{ } j_s \text{ is locally Lipschitz;} \\ (b) |\boldsymbol{\eta}| \leq S c(1 + |\boldsymbol{\xi}|), \quad \forall \boldsymbol{\xi} \in \mathbb{R}^d, \boldsymbol{\eta} \in \partial j_s(\boldsymbol{\xi}); \\ (c) j_s^0(\boldsymbol{\xi}; -\boldsymbol{\xi}) \leq S d(1 + |\boldsymbol{\xi}|), \quad \forall \boldsymbol{\xi} \in \mathbb{R}^d. \end{array} \right.$$

Furthermore if the assumption $H(\mu)(d)$ holds, then we have

$$(d) (\boldsymbol{\eta}_1 - \boldsymbol{\eta}_2) \cdot (\boldsymbol{\xi}_1 - \boldsymbol{\xi}_2) \geq -S\lambda |\boldsymbol{\xi}_1 - \boldsymbol{\xi}_2|^2,$$

$$\forall \boldsymbol{\xi}_1, \boldsymbol{\xi}_2 \in \mathbb{R}^d, \boldsymbol{\eta}_i \in \partial j_s(\boldsymbol{\xi}_i), i = 1, 2.$$

Corollary 1. *If the functional $j_s : \mathbb{R}^d \rightarrow \mathbb{R}$ satisfies $H(j_s)(a)-(c)$, then*

$$j_s^0(\boldsymbol{\xi}; -\boldsymbol{\xi}) \leq S(c + d)|\boldsymbol{\xi}|, \quad \forall \boldsymbol{\xi} \in \mathbb{R}^d. \quad (16)$$

Proof. If $|\boldsymbol{\xi}| \leq d/c$, then we have

$$\begin{aligned} j_s^0(\boldsymbol{\xi}; -\boldsymbol{\xi}) &= \max_{\mathbf{v} \in \partial j_s(\boldsymbol{\xi})} \mathbf{v} \cdot (-\boldsymbol{\xi}) \\ &\leq \max_{\mathbf{v} \in \partial j_s(\boldsymbol{\xi})} |\mathbf{v}| |\boldsymbol{\xi}| \\ &\leq S c(1 + |\boldsymbol{\xi}|) |\boldsymbol{\xi}| \leq S(c + d) |\boldsymbol{\xi}|. \end{aligned}$$

If $|\boldsymbol{\xi}| \geq d/c$, then we have $1 \leq c/d|\boldsymbol{\xi}|$, so

$$j_s^0(\boldsymbol{\xi}; -\boldsymbol{\xi}) \leq S d(1 + |\boldsymbol{\xi}|) \leq S(c + d) |\boldsymbol{\xi}|. \quad \blacksquare$$

We now turn to the variational formulation for the mechanical problem \mathcal{P}_M . To this end, we introduce the closed subspace of H_1 defined by

$$V = \{\mathbf{v} \in H_1 \mid \mathbf{v} = \mathbf{0} \text{ on } \Gamma_1, v_n = 0 \text{ on } \Gamma_3\}.$$

Since $\text{meas}(\Gamma_1) > 0$, Korn's inequality holds, and thus there exists $C_K > 0$ which depends only on Ω and Γ_1 such that

$$\|\boldsymbol{\varepsilon}(\mathbf{v})\|_Q \geq C_K \|\mathbf{v}\|_{H_1} \quad \forall \mathbf{v} \in V. \quad (17)$$

A proof of Korn's inequality may be found in the work of Nečas and Hlaváček (1981, p. 79). On V , we consider the inner product given by

$$(\mathbf{u}, \mathbf{v})_V = (\boldsymbol{\varepsilon}(\mathbf{u}), \boldsymbol{\varepsilon}(\mathbf{v}))_Q, \quad \forall \mathbf{u}, \mathbf{v} \in V. \quad (18)$$

Let $\|\cdot\|$ be the associated norm, i.e.,

$$\|\mathbf{v}\| = \|\boldsymbol{\varepsilon}(\mathbf{v})\|_Q, \quad \forall \mathbf{v} \in V. \quad (19)$$

It follows from (17) and (19) that $\|\cdot\|_{H_1}$ and $\|\cdot\|$ are equivalent norms on V and therefore $(V, \|\cdot\|)$ is a real Hilbert space. The duality pairing between V and V^* is denoted by $\langle \cdot, \cdot \rangle$. Moreover, by the Sobolev trace theorem and (17) we have a constant C_0 depending only on the domain Ω, Γ_1 and Γ_3 such that

$$\|\mathbf{v}\|_{L^2(\Gamma_3)^d} \leq C_0 \|\mathbf{v}\|, \quad \forall \mathbf{v} \in V. \quad (20)$$

Next we define the operator $B : V \rightarrow V^*$ and the bilinear form $a : V \times V \rightarrow \mathbb{R}$ by

$$\langle B\mathbf{u}, \mathbf{v} \rangle = a(\mathbf{u}, \mathbf{v}) = (\boldsymbol{\varepsilon}\boldsymbol{\varepsilon}(\mathbf{u}), \boldsymbol{\varepsilon}(\mathbf{v}))_Q, \quad (21)$$

and the functional $J_s : V \rightarrow \mathbb{R}$ by

$$J_s(\mathbf{v}) = \int_{\Gamma_3} j_s(\mathbf{v}_\tau(x)) d\Gamma. \quad (22)$$

We also denote by \mathbf{f} the element of V^* given by

$$\langle \mathbf{f}, \mathbf{v} \rangle = \int_{\Omega} \mathbf{f}_0 \cdot \mathbf{v} dx + \int_{\Gamma_2} \mathbf{f}_2 \cdot \mathbf{v} d\Gamma \quad (23)$$

for all $\mathbf{v} \in V$. From (6), it follows that the integrals in (23) are well defined.

Proceeding in a standard way, we obtain the following variational formulation of the frictional Problem \mathcal{P}_M .

Problem 2. (\mathcal{P}_V^1) Find a displacement field $\mathbf{u} \in V$ and the friction density $\boldsymbol{\xi}_\tau \in L^2(\Gamma_3)^d$ such that

$$\langle B\mathbf{u} - \mathbf{f}, \mathbf{v} \rangle = \int_{\Gamma_3} \boldsymbol{\xi}_\tau \cdot \mathbf{v}_\tau d\Gamma, \quad \forall \mathbf{v} \in V \quad (24)$$

with

$$-\boldsymbol{\xi}_\tau \in \partial j_s(\mathbf{u}_\tau) \text{ a.e. on } \Gamma_3.$$

The above problem is called a boundary HemiVariational Inequality (HVI). Now we define an auxiliary problem.

Problem 3. (\mathcal{P}_V^2) Find a displacement field $\mathbf{u} \in V$ such that

$$\mathbf{f} - B\mathbf{u} \in \partial J_s(\mathbf{u}) \text{ in } V^*. \quad (25)$$

Remark 3. Basic properties of the Clarke subdifferential (see Clarke, 1983, Theorem 2.7.2) guarantee that each solution to Problem \mathcal{P}_V^2 also solves Problem \mathcal{P}_V^1 .

Now we formulate the existence theorem for Problem \mathcal{P}_V^1 .

Theorem 1. *If the assumptions $H(\mathcal{E})(a)-(c)$, $H(\mu)(a)-(c)$ and $H(f)$ hold, then Problem \mathcal{P}_V^1 has a solution.*

Remark 4. Due to the growth condition $H(j_s)$ (b) we have that $\xi_\tau \in L^2(\Gamma_3)^d$. This regularity is stronger than the typical regularity $\xi_\tau \in X_\tau^*$, where $X_\tau = \{v|_{\Gamma_3}, v \in V\}$ obtained by Khenous *et al.* (2006a; 2006b).

Since the proof of Theorem 1 is based on the standard technique (cf. Naniewicz and Panagiotopoulos, 1995), we present only a short outline. First, we formulate the following lemmas, which can be easily proved.

Lemma 2. *If the assumptions $H(\mathcal{E})(a)-(c)$ hold, then the operator $B \in \mathcal{L}(V, V^*)$ is symmetric ($\forall u, v \in V \langle Bu, v \rangle = \langle Bv, u \rangle$) and coercive ($\forall u \in V \langle Bu, u \rangle \geq m\|u\|^2$ with $m > 0$).*

Lemma 3. *If the assumptions $H(\mu)(a)-(c)$ hold, then the functional J_s defined by (22) satisfies the following:*

- $$\left\{ \begin{array}{l} (a) J_s \text{ is well defined and locally Lipschitz;} \\ (b) \|\eta\|_{V^*} \leq C(1 + \|u\|), \forall u \in V, \eta \in \partial J_s(u) \\ \quad \text{with a constant } C > 0; \\ (c) J_s^0(u; -u) \leq D(1 + \|u\|), \forall u \in V \\ \quad \text{with a constant } D > 0. \end{array} \right.$$

We now turn to the proof of Theorem 1.

Proof. Due to Remark 3 it is enough to prove the existence of a solution to Problem \mathcal{P}_V^2 . To this end, we formulate it in the equivalent way: Find $v \in V$ such that

$$Tu \ni f, \tag{26}$$

where $T : V \rightarrow 2^{V^*}$ is a multivalued operator defined by

$$T(\cdot) = B + \partial J_s(\cdot). \tag{27}$$

We need to verify that the operator T is coercive and pseudomonotone and exploit the surjectivity result of Brezis (see, e.g., Denkowski *et al.*, 2003, Theorem 1.3.70). The coercivity of T follows from the condition (c) of Lemma 3 and the coercivity of B . We observe that ∂J_s is generalized pseudomonotone (see Clarke, 1983, Proposition 2.1.5(b); Denkowski *et al.*, 2003, Definition 1.3.63). Furthermore, ∂J_s has nonempty, convex and closed values (see Clarke, 1983, Proposition 2.1.2(a)) and, by the condition (b) in Lemma 3, it is bounded. By Theorem 1.3.66 of Denkowski *et al.* (2003), ∂J_s is pseudomonotone. Linearity, continuity and coercivity of B imply its pseudomonotonicity. Thus T is pseudomonotone, which completes the proof. ■

Now we pass to the uniqueness result for Problem \mathcal{P}_V^1 .

Theorem 2. *If the assumptions $H(\mathcal{E})(a)-(c)$, $H(\mu)(a)-(d)$, $H(f)$ hold and*

$$m > S\lambda C_0^2, \tag{28}$$

then the solution to Problem \mathcal{P}_V^1 is unique and Problems \mathcal{P}_V^1 and \mathcal{P}_V^2 are equivalent.

Proof. Let $(u^1, \xi_\tau^1), (u^2, \xi_\tau^2)$ solve Problem \mathcal{P}_V^1 . We consider Eqn. (24) for both solutions separately. Subtracting the equations from each other we get

$$\langle B(u^1 - u^2), v \rangle = \int_{\Gamma_3} (\xi_\tau^1 - \xi_\tau^2) \cdot v_\tau \, d\Gamma, \quad \forall v \in V. \tag{29}$$

Let us take $v = u^1 - u^2$ in (29). By the condition $H(\mathcal{E})(c)$, Lemma 2 and Lemma 1(d) as well as (20), we obtain

$$m\|u^1 - u^2\|^2 \leq S\lambda C_0^2 \|u^1 - u^2\|^2, \tag{30}$$

which implies that $\|u^1 - u^2\| = 0$ if (28) holds. From (29) it follows that $\int_{\Gamma_3} (\xi_\tau^1 - \xi_\tau^2) \cdot v_\tau \, d\Gamma = 0$. The equality $\xi_\tau^1 = \xi_\tau^2$ comes from the density of the traces of the elements of V in the space X_τ . ■

Let now $u \in V$ be the solution to Problem \mathcal{P}_V^1 and let σ be the stress field given by (9). By a standard procedure, it can be shown that

$$\text{Div } \sigma + f_0 = 0 \quad \text{a.e. in } \Omega. \tag{31}$$

Thus, $\sigma \in Q_1$. A pair of functions (u, σ) which satisfies (24) and (9) is called a *weak solution* of the bilateral contact problem with nonmonotone friction law (9)–(13). Thus Problem \mathcal{P}_M has a unique weak solution. Note that under the assumption

$$\sigma n \in L^2(\Gamma)^d \tag{32}$$

it can be further shown that

$$\sigma n = f_2 \quad \text{a.e. on } \Gamma_2 \tag{33a}$$

and

$$-\sigma_\tau \in \partial j_s(u_\tau) \quad \text{a.e. on } \Gamma_3. \tag{33b}$$

4. Galerkin approximation and the error estimate

In this section we present the numerical scheme for Problem \mathcal{P}_V^1 based on the Galerkin method as well as the estimates for the error of this scheme.

Let V^h be a finite dimensional linear subspace of V equipped with the norm of V , where $h > 0$ denotes

the spatial discretization parameter. In the numerical simulations presented in the next sections, V^h consists of continuous and piecewise affine functions, that is,

$$V^h = \{v^h \in [C(\bar{\Omega})]^d : \forall Tr \in \mathcal{T}^h \quad v^h|_{Tr} \in [P_1(Tr)]^d, \\ v^h = \mathbf{0} \text{ on } \Gamma_1, v_n^h = 0 \text{ on } \Gamma_3\},$$

where Ω is assumed to be a polygonal domain, \mathcal{T}^h denotes a finite element triangulation of $\bar{\Omega}$, and $P_1(Tr)$ represents the space of polynomials of global degree less or equal to one in Tr . The discrete approximation of Problem \mathcal{P}_V^1 takes the following form

Problem 4. (\mathcal{P}_V^h) Find a displacement field $\mathbf{u}^h \in V^h$ and the friction density $\xi_\tau^h \in L^2(\Gamma_3)^d$ such that

$$\langle B\mathbf{u}^h - \mathbf{f}, \mathbf{v}^h \rangle = \int_{\Gamma_3} \xi_\tau^h \cdot \mathbf{v}_\tau^h \, d\Gamma, \quad \forall \mathbf{v}^h \in V^h \quad (34)$$

with

$$-\xi_\tau^h \in \partial j_s(\mathbf{u}_\tau^h) \quad \text{a.e. on } \Gamma_3.$$

The existence and uniqueness of solutions to this problem can be proved analogously to those for Problem \mathcal{P}_V^1 . We now formulate a result which estimates the error between the solutions to Problems \mathcal{P}_V^1 and \mathcal{P}_V^h .

Theorem 3. Let $\mathbf{u} \in V$ and $\mathbf{u}^h \in V^h$ solve Problems \mathcal{P}_V^1 and \mathcal{P}_V^h , respectively. If the assumptions $H(\mathcal{E})(a)-(c)$, $H(\mu)(a)-(d)$, $H(f)$ and (28) hold, then there exists a constant $C_1 > 0$, depending only on the given data, such that

$$\|\mathbf{u} - \mathbf{u}^h\| \leq C_1 \sqrt{r}, \quad (35)$$

where $r = \inf_{\mathbf{v}^h \in V^h} \|\mathbf{u} - \mathbf{v}^h\|$.

Proof. Let $\mathbf{u} \in V$ and $\mathbf{u}^h \in V^h$ solve \mathcal{P}_V^1 and \mathcal{P}_V^h , respectively. Let us take ξ_τ and ξ_τ^h for which the pairs (\mathbf{u}, ξ_τ) and $(\mathbf{u}^h, \xi_\tau^h)$ satisfy the relations (24) and (34), respectively. First we estimate the norm of the solution \mathbf{u} . To this end, we take $\mathbf{v} = \mathbf{u}$ in (24) and by Lemma 2, the definition of the Clarke subdifferential, Corollary 1 and the Hölder inequality we obtain

$$m\|\mathbf{u}\|^2 \leq \langle B\mathbf{u}, \mathbf{u} \rangle = \langle f, \mathbf{u} \rangle + \int_{\Gamma_3} -\xi_\tau \cdot (-\mathbf{u}_\tau) \, d\Gamma \\ \leq \|f\|_{V^*} \|\mathbf{u}\| + S(c+d) \int_{\Gamma_3} |\mathbf{u}_\tau| \, d\Gamma \\ \leq \|\mathbf{u}\| \left(\|f\|_{V^*} + S(c+d)C_0 \sqrt{\text{meas}(\Gamma_3)} \right), \quad (36)$$

where $\text{meas}(\Gamma_3)$ denotes the Lebesgue measure of the set Γ_3 . From (36) we get

$$\|\mathbf{u}\| \leq \alpha := \frac{\|f\|_{V^*} + S(c+d)C_0 \sqrt{\text{meas}(\Gamma_3)}}{m}. \quad (37)$$

Now we estimate the norm of ξ_τ by means of $\|\mathbf{u}\|$. Integrating the inequality $H(j_s)(b)$ over Γ_3 we obtain by simple calculations

$$\|\xi_\tau\|_{L^2(\Gamma_3)^d} \\ \leq Sc\sqrt{2\text{meas}(\Gamma_3)} + \sqrt{2}ScC_0\|\mathbf{u}\| := \beta + \delta\|\mathbf{u}\|. \quad (38)$$

We can obtain an analogous estimate for \mathbf{u}^h and ξ_τ^h :

$$\|\mathbf{u}^h\| \leq \alpha, \quad \|\xi_\tau^h\|_{L^2(\Gamma_3)^d} \leq \beta + \delta\|\mathbf{u}^h\|. \quad (39)$$

Since $V^h \subset V$, we observe that for each $\mathbf{v}^h \in V^h$ both (24) and (34) hold. We obtain

$$\langle B(\mathbf{u} - \mathbf{u}^h), \mathbf{v}^h \rangle - \int_{\Gamma_3} (\xi_\tau(x) - \xi_\tau^h(x)) \cdot \mathbf{v}_\tau^h(x) \, d\Gamma = 0, \quad \forall \mathbf{v}^h \in V^h. \quad (40)$$

By Lemma 2 we have

$$m\|\mathbf{u} - \mathbf{u}^h\|^2 \leq \langle B(\mathbf{u} - \mathbf{u}^h), \mathbf{u} - \mathbf{u}^h \rangle, \quad (41)$$

and by the condition $H(j_s)(d)$ and (20), after integration over Γ_3 we get

$$-S\lambda C_0^2 \|\mathbf{u} - \mathbf{u}^h\|^2 \\ \leq - \int_{\Gamma_3} (\xi_\tau(x) - \xi_\tau^h(x)) \cdot (\mathbf{u}_\tau(x) - \mathbf{u}_\tau^h(x)) \, d\Gamma. \quad (42)$$

From (41) and (42) we obtain for all $\mathbf{v}^h \in V^h$

$$(m - S\lambda C_0^2) \|\mathbf{u} - \mathbf{u}^h\|^2 \\ \leq \langle B(\mathbf{u} - \mathbf{u}^h), \mathbf{u} - \mathbf{u}^h \rangle \\ - \int_{\Gamma_3} (\xi_\tau(x) - \xi_\tau^h(x)) \cdot (\mathbf{u}_\tau(x) - \mathbf{u}_\tau^h(x)) \, d\Gamma \\ = \langle B(\mathbf{u} - \mathbf{u}^h), \mathbf{u} - \mathbf{v}^h \rangle \\ - \int_{\Gamma_3} (\xi_\tau(x) - \xi_\tau^h(x)) \cdot (\mathbf{u}_\tau(x) - \mathbf{v}_\tau^h(x)) \, d\Gamma \\ \leq \|B\|_{\mathcal{L}(V, V^*)} (\|\mathbf{u}\| + \|\mathbf{u}^h\|) \|\mathbf{u} - \mathbf{v}^h\| \\ + (\|\xi_\tau\|_{L^2(\Gamma_3)^d} + \|\xi_\tau^h\|_{L^2(\Gamma_3)^d}) C_0 \|\mathbf{u} - \mathbf{v}^h\|. \quad (43)$$

The equality in (43) is a consequence of the application of (40) with $\mathbf{v}^h = \mathbf{u}^h$. Combining (37)–(39) and (43), we obtain for all $\mathbf{v}^h \in V^h$

$$(m - S\lambda C_0^2) \|\mathbf{u} - \mathbf{u}^h\|^2 \\ \leq \|B\|_{\mathcal{L}(V, V^*)} 2\alpha \|\mathbf{u} - \mathbf{v}^h\| + \kappa \|\mathbf{u} - \mathbf{v}^h\|, \quad (44)$$

where the constant $\kappa > 0$ depends only on the data of the problem. Rearranging the last inequality and using the fact that $\mathbf{v}^h \in V^h$ is arbitrary, we conclude the proof. ■

Remark 5. Theorem 3 means that the error of the Galerkin method has the order $O(\sqrt{\inf_{\mathbf{v}^h \in V^h} \|\mathbf{u} - \mathbf{v}^h\|})$.

Remark 6. In the above case an estimation of the error between ξ and ξ^h is an open problem. To obtain such estimates, a discrete Babuska–Brezzi inf-sup condition is used. Such a condition requires the discretization of the contact boundary stress (cf. Hild and Renard, 2007).

5. Nonconvex proximal bundle method

In this section we use a numerical strategy of nonconvex and nonsmooth optimization known as the proximal bundle method to solve the Galerkin problem \mathcal{P}_V^h . To that end, we first need to reformulate it as a minimization problem in \mathbb{R}^N .

Let us consider a basis $\mathbb{B} = \{\mathbf{v}_1^h, \dots, \mathbf{v}_N^h\} \subset V^h$ of the space V^h , where $N = \dim V^h$, and define the function $h : \mathbb{R}^N \rightarrow \mathbb{R}$ by

$$h(\alpha) = \int_{\Gamma_3} j_s \left(\sum_{j=1}^N \alpha_j \mathbf{v}_j^h(x) \right) d\Gamma, \quad \forall \alpha \in \mathbb{R}^N. \quad (45)$$

We observe (cf. Clarke, 1983, Theorems 2.3.10 and 2.7.2) that if $\eta \in \partial h(\alpha)$ then there exists $\xi_\tau^h \in L^2(\Gamma_3)^d$ such that $-\xi_\tau^h(x) \in \partial j_s \left(\sum_{j=1}^N \alpha_j \mathbf{v}_j^h(x) \right)$ a.e. on Γ_3 and

$$\begin{aligned} & \langle \eta, \zeta \rangle_{\mathbb{R}^N \times \mathbb{R}^N} \\ &= \int_{\Gamma_3} -\xi_\tau^h(x) \cdot \left(\sum_{j=1}^N \zeta_j \mathbf{v}_j^h(x) \right) d\Gamma, \quad \forall \zeta \in \mathbb{R}^N. \end{aligned} \quad (46)$$

From this observation we conclude that Problem \mathcal{P}_V^h can be replaced by the following.

Problem 5. ($\mathcal{P}_{\mathbb{R}^N}^h$) Find $\alpha \in \mathbb{R}^N$ such that

$$\mathcal{F} - \alpha^T \mathcal{B} \in \partial h(\alpha), \quad (47)$$

where

$$\begin{aligned} \mathcal{F} &= [F_i]_{i=1}^N, & F_i &= \langle f, \mathbf{v}_i^h \rangle, \\ \mathcal{B} &= [B_{ij}]_{ij=1}^N, & B_{ij} &= \langle B \mathbf{v}_i^h, \mathbf{v}_j^h \rangle. \end{aligned}$$

In fact, if $\bar{\alpha} \in \mathbb{R}^N$ satisfies (47), then $\bar{u} = \sum_{j=1}^N \bar{\alpha}_j \mathbf{v}_j^h$ solves Problem \mathcal{P}_V^h . By Proposition 2.3.3 of Clarke (1983), the inclusion (47) is equivalent to the following:

$$0 \in \partial H(\alpha),$$

where

$$H(\alpha) = \frac{1}{2} \alpha^T \mathcal{B} \alpha - \mathcal{F}^T \alpha + h(\alpha). \quad (48)$$

In the sequel we will use the fact that if $\alpha \in \mathbb{R}^N$ is a local minimizer (or maximizer) of the functional H then it satisfies (48) (cf. Clarke, 1983, Proposition 2.3.2).

Remark 7. If the assumptions $H(\mathcal{E})(a)–(c)$, $H(f)$, $H(\mu)(a)–(d)$ as well as (28) hold, then the functional H is convex and convex programming algorithms can be used for its minimization. If we omit $H(\mu)(d)$ and (28), then the solution is not necessarily unique and there is no error estimate provided by Theorem 3, only the strong convergence of a subsequence of Galerkin solutions to the exact solution can be proved (see Haslinger *et al.*, 1999, Theorem 3.4). The nonconvex PBM, however, remains still valid in such a case.

Now we formulate the lemma on the existence of a global minimizer of H .

Lemma 4. Under the assumptions $H(\mathcal{E})(a)–(c)$, $H(f)$ and $H(\mu)(a)–(c)$, the functional H defined in (48) attains a global minimum.

Proof. It is enough to verify that the functional H is proper, lower semicontinuous and coercive. The fact that H is proper is obvious and the lower semi-continuity follows from the fact that h is locally Lipschitz. For coercivity we need to estimate $H(\alpha)$ from below. Let $\alpha \in \mathbb{R}^N$ and $\mathbf{v} = \sum_{i=1}^N \alpha_i \mathbf{v}_i^h$. Then by means of Lemma 2 and the Lebourg mean-value Theorem (see Clarke, 1983, Theorem 2.3.7) we obtain

$$\begin{aligned} H(\alpha) &= \frac{1}{2} \langle B \mathbf{v}, \mathbf{v} \rangle - \langle \mathbf{f}, \mathbf{v} \rangle + \int_{\Gamma_3} j_s(\mathbf{v}_\tau(x)) - j_s(0) d\Gamma \\ &\quad + \int_{\Gamma_3} j_s(0) d\Gamma \\ &\geq \frac{1}{2} m \|\mathbf{v}\|^2 - \|\mathbf{f}\|_{V^*} \|\mathbf{v}\| \\ &\quad - \int_{\Gamma_3} \eta(x) \cdot (-\mathbf{v}_\tau(x)) d\Gamma + j_s(0) \text{meas}(\Gamma_3), \end{aligned} \quad (49)$$

where $\eta(x) \in \partial j_s(\theta(x) \mathbf{v}_\tau(x))$ a.e. on Γ_3 with $\theta(x) \in (0, 1)$. Furthermore, using the conditions $H(j)(b)–(c)$, Corollary 1 and the Cauchy formula with ε , we get

$$\begin{aligned} & \int_{\Gamma_3} \eta(x) \cdot (-\mathbf{v}_\tau(x)) d\Gamma \\ &= \int_{\Gamma_3} \frac{1}{\theta(x)} \eta(x) \cdot (-\theta(x) \mathbf{v}_\tau(x)) d\Gamma \\ &\leq \int_{\Gamma_3} \frac{1}{\theta(x)} j_s^0(\theta(x) \mathbf{v}_\tau(x); \\ &\quad -\theta(x) \mathbf{v}_\tau(x)) d\Gamma \\ &\leq \int_{\Gamma_3} S(c+d) |\mathbf{v}_\tau(x)| d\Gamma \\ &\leq \frac{S^2(c+d)^2 m(\Gamma_3)}{2\varepsilon} - \frac{\varepsilon}{2} C_0^2 \|\mathbf{v}\|^2, \quad \forall \varepsilon > 0. \end{aligned} \quad (50)$$

Using (49), (50) with $\varepsilon < m/C_0^2$ and the fact that the expression $\rho(\alpha) = \|\sum_{i=1}^N \alpha_i \mathbf{v}_i^h\|$ is a well defined norm on \mathbb{R}^N , we conclude that H is coercive. ■

Now we turn to the formulation of the proximal bundle algorithm for nonconvex optimization used to minimize the function H . Its scheme is given as Algorithm 1. We refer to the works of Mäkelä (1990; 2001), Mäkelä *et al.* (1999) as well as Haslinger *et al.* (1999) and the references therein for details.

Algorithm 1. Proximal bundle algorithm.

Let $\varepsilon > 0$ and $\mathbf{x}_1 = \mathbf{y}_1 \in \mathbb{R}^N$ be given.

Let $\mathbf{g}_1 \in \partial H(\mathbf{y}_1)$, $k = 1$, $v_1 = -\infty$.

While $v_k < -\varepsilon$, do

Step 1. DIRECTION FINDING.

Find direction and descent ratio v_k .

Step 2. LINE SEARCH.

Find \mathbf{x}_{k+1} and \mathbf{y}_{k+1} .

Step 3. Take $\mathbf{g}_{k+1} \in \partial H(\mathbf{y}_{k+1})$, $k = k + 1$.

End do

The algorithm constructs sequences (\mathbf{x}_k) and (\mathbf{y}_k) . The starting point \mathbf{x}_1 is obtained as a solution to a frictionless problem, although it can be chosen arbitrarily. The sequence (\mathbf{x}_k) is supposed to converge to a local minimum of H , while the auxiliary sequence (\mathbf{y}_k) is used to construct the function \hat{H}_k , being a piecewise linear approximation of H . The search direction \mathbf{d}_k is obtained as a solution to the problem

$$\text{minimize } \hat{H}_k(\mathbf{x}_k + \mathbf{d}) + \frac{1}{2} u_k \mathbf{d}^T \mathbf{d}, \quad \text{over } \mathbf{d} \in \mathbb{R}^N, \quad (51)$$

where the function \hat{H}_k is defined as in the formulas (4), (5), (7), (8) by Mäkelä *et al.* (1999) and u_k is an arbitrarily chosen weight. The problem (51) is a smooth quadratic programming problem with linear inequality constraints, which can be solved by standard quadratic optimization techniques. The value v_k is the predicted amount of descent given by

$$v_k = \hat{H}_k(\mathbf{x}_k + \mathbf{d}_k) - H(\mathbf{x}_k).$$

The point \mathbf{x}_{k+1} is obtained in the line search procedure as $\mathbf{x}_k + t_L^k \mathbf{d}_k$, where $t_L^k \in [0, 1]$ is the largest number such that $H(\mathbf{x}_{k+1}) \leq H(\mathbf{x}_k) + m_L t_L^k v_k$ with $m_L \in (0, 1/2)$ being an arbitrary parameter. Depending on the fact whether t_L^k is greater or less than the arbitrary parameter $\bar{t} \in (0, 1]$, either $\mathbf{y}_{k+1} = \mathbf{x}_{k+1}$ (the so-called long step) or $\mathbf{y}_{k+1} = \mathbf{x}_k + t_R^k \mathbf{d}_k$ with $t_R^k > t_L^k$ is given according to the formula (12) by Mäkelä *et al.* (1999) (the so-called short step). The idea of the short step is that, since we are in the vicinity of the nonsmoothness of H , its value is not decreasing significantly in the descent direction \mathbf{d}_k . Then we enrich the piecewise linear approximation by taking \mathbf{y}_{k+1} on the other side of the nonsmoothness.

The Schur complement method is used for the compression of unknowns (see Haslinger *et al.*, 1999,

Chapter 6.1), while for the solution of the problem (51) we use the LIBQP library (see Franc, 2011). For calculation of values of $h(\alpha)$ and an element of $\partial h(\alpha)$, the trapezoidal quadrature rule was used.

6. Solution based on a sequence of convex programming problems

The numerical strategy presented in this section is based on a sequence of convex programming problems; more details can be found in the works of Mistakidis and Panagiotopoulos (1997; 1998). This approach is implemented by using an iterative procedure in which for each iteration the friction coefficient μ is fixed to a given function depending on the tangential displacement solution \mathbf{u}_τ found in the previous iteration. Then, the nonsmooth convex problems arising during the iterative process can be solved by classical numerical methods. In the following, we consider the discrete space $Y_\tau^h \subset L^2(\Gamma_3)^d$ related to the discretization of the friction density σ_τ . We also consider the boundary interpolation operator $\Pi_h : V_h \rightarrow Y_\tau^h$ (see the works of Khenous *et al.* (2006a; 2006b) for more details about the discretization spaces). The numerical solution of the nonsmooth nonconvex variational problem \mathcal{P}_V^h is based on the iterative scheme given as Algorithm 2.

Algorithm 2. Numerical solution of Problem \mathcal{P}_V^h .

Let $\epsilon > 0$ and $\mathbf{u}^{(0)}$ be given.

Then for $k = 0, 1, \dots$,

Problem $\mathcal{P}_{V_C}^h$. Find a displacement field $\mathbf{u}^{h,(k+1)} \in V^h$ and a friction stress field $\sigma_\tau^{h,(k+1)} \in Y_\tau^h$ such that

$$\begin{aligned} & \langle B\mathbf{u}^{h,(k+1)} - \mathbf{f}, \mathbf{v}^h \rangle \\ & = \int_{\Gamma_3} \sigma_\tau^{h,(k+1)} \cdot \mathbf{v}_\tau^h \, d\Gamma, \quad \forall \mathbf{v}^h \in V^h \end{aligned} \quad (52)$$

with

$$-\sigma_\tau^{h,(k+1)} \in \mu(|\Pi_h \mathbf{u}_\tau^{h,(k)}|) S\partial|\Pi_h \mathbf{u}_\tau^{h,(k+1)}| \text{ on } \Gamma_3 \quad (53)$$

until $\|\mathbf{u}^{h,(k+1)} - \mathbf{u}^{h,(k)}\| \leq \epsilon \|\mathbf{u}^{h,(k)}\|$
 and $\|\sigma_\tau^{h,(k+1)} - \sigma_\tau^{h,(k)}\|_{L^2(\Gamma_3)^d} \leq \epsilon \|\sigma_\tau^{h,(k)}\|_{L^2(\Gamma_3)^d}$

Here, k represents the index of the iterative procedure and (53) is derived from (15) using (14). This numerical strategy leads to the solution of a nonsmooth convex problem $\mathcal{P}_{V_C}^h$ at each iteration k . See Fig. 1 for a graphical representation of the iterative algorithm at a contact node.

In the rest of the section, to simplify the readability, we skip the dependence on the iteration index k . In Problem $\mathcal{P}_{V_C}^h$ the discrete stress σ_τ^h on the contact

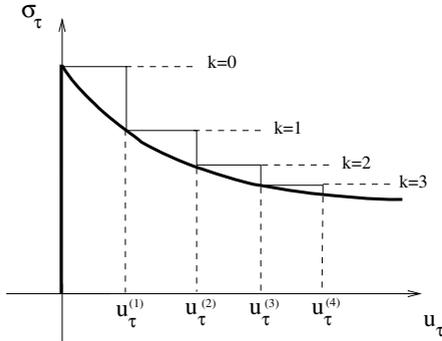


Fig. 1. Graphical description of the iterative algorithm at a contact node.

boundary Γ_3 can be viewed as a Lagrange stress multiplier.

The unique solvability of Problem $\mathcal{P}_{V_C}^h$ follows from arguments of variational inequalities and a fixed point theorem, similar to those used by Barbotou *et al.* (2002) as well as Han and Sofonea (2002). Moreover, the numerical analysis of this problem (including error estimates and convergence results) can be provided by extending the arguments already used by Barbotou *et al.* (2002) as well as Han and Sofonea (2002). Nevertheless, to keep this paper at a reasonable length, we skip this analysis and we pass in what follows to a brief description of the numerical algorithm used to solve Problem $\mathcal{P}_{V_C}^h$.

For the numerical treatment of the nonsmooth convex Problem $\mathcal{P}_{V_C}^h$ we use the augmented Lagrangean approach. To this end we consider additional fictitious nodes for the Lagrange multiplier in the initial mesh. The construction of these nodes depends on the contact element used for the geometrical discretization of the interface Γ_3 . In the case of the numerical example presented in Section 7, the discretization is based on the “node-to-rigid” contact element, which is composed by one node of Γ_3 and one Lagrange multiplier node. This contact interface discretization is characterized by a finite dimensional subspace $H_{\Gamma_3}^h \subset Y_\tau^h$. Let N_{tot} be the total number of nodes and denote by α^i the basis functions of the spaces V^h for $i = 1, \dots, N_{tot}$. Moreover, let N_{Γ_3} represent the number of nodes on the interface Γ_3 and let μ^i be the shape functions of the finite element space $H_{\Gamma_3}^h$, for $i = 1, \dots, N_{\Gamma_3}$; thus,

$$H_{\Gamma_3}^h = \{ \gamma^h \in Y_\tau^h : \gamma^h = \sum_{i=1}^{N_{\Gamma_3}} \gamma^i \mu^i \}.$$

Usually, if a P_1 finite element method is used for the displacement, then a P_0 finite element method is considered for the multipliers (see Khenous *et al.*, 2006a; 2006b). Then, the expression of functions $v^h \in V^h$ and

$\gamma^h \in H_{\Gamma_3}^h$ is given by

$$v^h = \sum_{i=1}^{N_{tot}} w^i \alpha^i, \quad \forall v^h \in V^h,$$

$$\gamma^h = \sum_{i=1}^{N_{\Gamma_3}} \gamma^i \mu^i, \quad \forall \gamma^h \in H_{\Gamma_3}^h,$$

where v^i represent the values of the corresponding functions v^h at the i -th node of \mathcal{T}^h . Also, γ^i denotes the values of the function γ^h at the i -th node of the contact element discretization of the contact interface. More details about this discretization step can be found in the works of Alart and Curnier (1991), Khenous *et al.* (2006b), as well as Wriggers (2002).

The augmented Lagrangean approach we use shows that Problem $\mathcal{P}_{V_C}^h$ can be governed by a system of nonlinear equations of the following form.

Problem 6. (\mathcal{P}_L) Find a displacement field $\mathbf{u} \in \mathbb{R}^{d \cdot N_{tot}}$, a stress multiplier field $\boldsymbol{\lambda} \in \mathbb{R}^{d \cdot N_{\Gamma_3}}$ such that

$$\mathbf{R}(\mathbf{u}, \boldsymbol{\lambda}) = \tilde{\mathbf{G}}(\mathbf{u}) + \mathcal{F}(\mathbf{u}, \boldsymbol{\lambda}) = \mathbf{0}, \quad (54)$$

where $\tilde{\mathbf{G}}$ and \mathcal{F} are defined below.

A brief description of the notation used in this problem is the following.

First, the vectors, \mathbf{u} and $\boldsymbol{\lambda}$ represent the displacement and the generalized Lagrange multiplier, respectively. They are defined by

$$\mathbf{u} = \{ \mathbf{u}^i \}_{i=1}^{N_{tot}}, \quad \boldsymbol{\lambda} = \{ \boldsymbol{\lambda}^i \}_{i=1}^{N_{\Gamma_3}}, \quad (55)$$

where \mathbf{u}^i represents the value of the corresponding function \mathbf{u}^h at the i -th node of \mathcal{T}^h . Also, $\boldsymbol{\lambda}^i$ denotes the value of the corresponding function $\boldsymbol{\lambda}^h$ at the i -th node of the contact element of the discretized contact interface.

In addition, the generalized elastic term $\tilde{\mathbf{G}}(\mathbf{u}) \in \mathbb{R}^{d \cdot N_{tot}} \times \mathbb{R}^{d \cdot N_{\Gamma_3}}$ is defined by $\tilde{\mathbf{G}}(\mathbf{u}) = (\mathbf{G}(\mathbf{u}), \mathbf{0}_{d \cdot N_{\Gamma_3}})$. Here $\mathbf{0}_{d \cdot N_{\Gamma_3}}$ is the zero element of $\mathbb{R}^{d \cdot N_{\Gamma_3}}$; also, $\mathbf{G}(\mathbf{u}) \in \mathbb{R}^{d \cdot N_{tot}}$ denotes the elastic term, respectively, given by

$$(\mathbf{G}(\mathbf{u}) \cdot (\mathbf{v}))_{\mathbb{R}^{d \times N_{tot}}} = \langle B\mathbf{u}^h - \mathbf{f}, \mathbf{v}^h \rangle, \quad \forall \mathbf{u}^h, \mathbf{v}^h \in V^h.$$

Above, \mathbf{u} and \mathbf{v} represent the generalized vectors of coordinates \mathbf{u}^i and \mathbf{v}^i for $i = 1, \dots, N_{tot}$, respectively, and note that the volume and surface efforts are contained in the term $\mathbf{G}(\mathbf{u})$.

The contact operator $\mathcal{F}(\mathbf{u}, \boldsymbol{\lambda})$, which allows dealing with the friction law, is given by

$$(\mathcal{F}(\mathbf{u}, \boldsymbol{\lambda}) \cdot (\mathbf{v}, \boldsymbol{\gamma}))_{\mathbb{R}^{d \cdot N_{tot}} \times \mathbb{R}^{d \cdot N_{\Gamma_3}}}$$

$$= \int_{\Gamma_3} \nabla \mathbf{u} l_\tau^r(\mathbf{u}^h, \boldsymbol{\lambda}^h) \cdot \mathbf{v}^h \, d\Gamma$$

$$+ \int_{\Gamma_3} \nabla \boldsymbol{\lambda} l_\tau^r(\mathbf{u}^h, \boldsymbol{\lambda}^h) \cdot \boldsymbol{\gamma}^h \, d\Gamma,$$

$$\forall \mathbf{u}, \mathbf{v} \in \mathbb{R}^{d \cdot N_{tot}}, \forall \boldsymbol{\lambda}, \boldsymbol{\gamma} \in \mathbb{R}^{d \cdot N_{\Gamma_3}},$$

$$\forall \mathbf{u}^h, \mathbf{v}^h \in V^h, \forall \boldsymbol{\lambda}^h, \boldsymbol{\gamma}^h \in H_{\Gamma_3}^h.$$

Here the Lagrangean multiplier λ and its virtual variable γ represent the friction forces. Moreover, l_τ^r denotes the augmented Lagrangean functional given by

$$l_\tau^r(\mathbf{u}^h, \lambda^h) = \mathbf{u}_\tau^h \cdot \lambda^h + \frac{r}{2} |\mathbf{u}_\tau^h|^2 - \frac{1}{2r} (\text{dist}_{C[-\mu S]} \{ \lambda^h + r \mathbf{u}_\tau^h \})^2, \quad (56)$$

where r is a positive penalty coefficient and the Coulomb convex set $C[-\mu S]$ denotes the convex disk of radius $-\mu S$. For more details about the “quasi”-Lagrangean method, we refer the reader to the works of Alart and Curnier (1991) as well as Wriggers (2002).

The solution of the nonlinear system (54) is based on a linear iterative technique similar to that used in the Newton method. The latter permits to treat both variables (\mathbf{u}, λ) simultaneously. Then, we consider the pair $\tilde{\mathbf{u}} = (\mathbf{u}, \lambda)$ and the iterative scheme can be summarized as Algorithm 3.

Algorithm 3. Linear scheme based on the Newton algorithm.

Let $\epsilon > 0$ and $\tilde{\mathbf{u}}_0$ be given.

For $i = 0, \dots$ compute

$$\tilde{\mathbf{u}}_{i+1} = (\mathbf{K}_i + \mathbf{T}_i)^{-1} \cdot \mathbf{R}(\tilde{\mathbf{u}}_i) \quad (57)$$

until $\|\tilde{\mathbf{u}}_{i+1} - \tilde{\mathbf{u}}_i\|_{\mathbb{R}^{d \cdot N_{tot} + d \cdot N_{\Gamma_3}}} \leq \epsilon \|\tilde{\mathbf{u}}_i\|_{\mathbb{R}^{d \cdot N_{tot} + d \cdot N_{\Gamma_3}}}$

and $\|\mathbf{R}(\tilde{\mathbf{u}}_{i+1}) - \mathbf{R}(\tilde{\mathbf{u}}_i)\|_{\mathbb{R}^{d \cdot N_{tot} + d \cdot N_{\Gamma_3}}} \leq \epsilon \|\mathbf{R}(\tilde{\mathbf{u}}_i)\|_{\mathbb{R}^{d \cdot N_{tot} + d \cdot N_{\Gamma_3}}}$

Here i represents the Newton iteration index, $\mathbf{K}_i = D_{\tilde{\mathbf{u}}} \mathbf{G}(\mathbf{u})$ denotes the usual elastic stiffness matrix and $\mathbf{T}_i \in \partial_{\tilde{\mathbf{u}}} \mathcal{F}(\tilde{\mathbf{u}}_i)$ is the frictional tangent matrix. $D_{\tilde{\mathbf{u}}} \mathbf{G}$ represents the differential of the functions \mathbf{G} with respect to the variable $\tilde{\mathbf{u}}$. $\partial_{\tilde{\mathbf{u}}} \mathcal{F}(\tilde{\mathbf{u}}_i)$ represents the generalized Jacobian of \mathcal{F} at $\tilde{\mathbf{u}}_i$. Usually $\tilde{\mathbf{u}}_i$ is in a region of linearity since the point set of nondifferentiability of the function \mathcal{F} has null measure. Then \mathbf{T}_i is reduced to a single classical Jacobian matrix. Each region of differentiability of the operator \mathcal{F} corresponds to a friction state of each contact node of the discretization.

It is easy to see that (57) is equivalent to an algebraic linear system which can be solved by a conjugate gradient method with efficient preconditioners, used here to overcome the poor conditioning of the matrix due to the contact terms, see for instance, Alart et al. (1997). Details on computational contact mechanics, including algorithms similar to that described above, can be found in the monographs of Laursen (2002) and Wriggers (2002). Finally, recall that a similar numerical approach, in the study of contact problems with piezoelectric materials, is presented by Barboteu and Sofonea (2009). Details on

these the classical algorithms can be found in the work of Alart and Curnier (1991), as well as Wriggers (2002).

7. Numerical examples

The main purpose of this section is to validate the numerical solution based on the PBM presented in Section 5 in comparison with the more standard method presented in Section 6. In order to do that, we consider an academic two-dimensional example of Problem \mathcal{P}_M whose physical setting is depicted in Fig. 2.

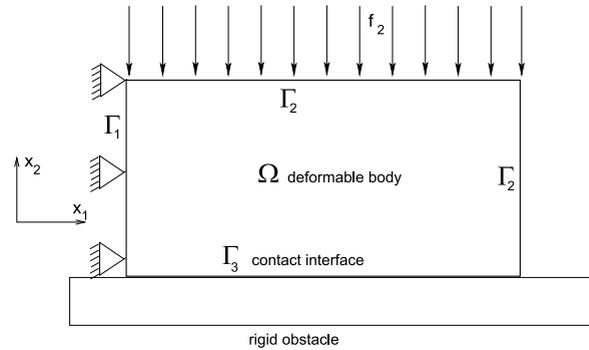


Fig. 2. Initial configuration of the two-dimensional example.

Let $\Omega = (0, L_1) \times (0, L_2) \subset \mathbb{R}^d$, $L_1, L_2 > 0$ be a rectangle with boundary Γ . We divide Γ into three regions:

$$\Gamma_3 = [0, L_1] \times \{0\},$$

$$\Gamma_1 = \{0\} \times [0, L_2],$$

$$\Gamma_2 = \Gamma \setminus (\Gamma_1 \cup \Gamma_3).$$

There, we consider the domain Ω as the cross section of a three-dimensional linearly elastic body subjected to the action of tractions in such a way that a plane stress hypothesis is assumed. On the part $\Gamma_1 = \{0\} \times [0, L_2]$ the body is clamped and therefore the displacement field vanishes there. Vertical tractions act on the part $[0, L_1] \times \{L_2\}$ of the boundary, and the part $\{L_1\} \times [0, L_2]$ is traction free. Thus, $\Gamma_2 = (\{L_1\} \times [0, L_2]) \cup ([0, L_1] \times \{L_2\})$. No body forces are assumed to act on the elastic body during the process. The body is in bilateral frictional contact with a rigid obstacle on the part $\Gamma_3 = [0, L_1] \times \{0\}$ of the boundary. The friction is modeled by a nonmonotone law in which the friction coefficient μ depends on the tangential displacement $|\mathbf{u}_\tau|$. Let us consider the following friction bound function $\mu : \mathbb{R}^d \rightarrow \mathbb{R}$:

$$\mu(|\mathbf{u}_\tau|) = (a - b) \cdot e^{-\alpha |\mathbf{u}_\tau|} + b, \quad (58)$$

with $a, b, \alpha > 0$, $a \geq b$. See Fig. 3 for a representation of the friction bound function μ with the values $a = 0.004$, $b = 0.002$ and $\alpha = 100$ used in the simulations.

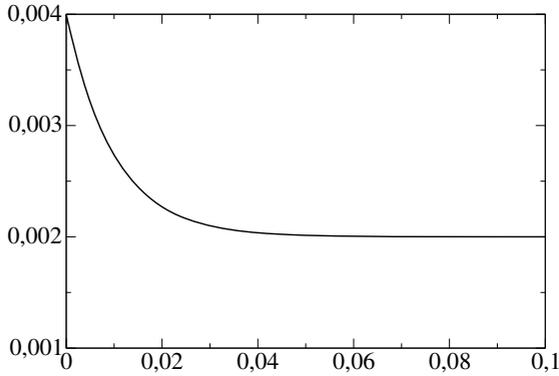


Fig. 3. Graph of the friction bound function μ .

The elasticity tensor \mathcal{E} satisfies

$$(\mathcal{E}\tau)_{\alpha\beta} = \frac{E\kappa}{1-\kappa^2}(\tau_{11} + \tau_{22})\delta_{\alpha\beta} + \frac{E}{1+\kappa}\tau_{\alpha\beta}, \quad (59)$$

with $1 \leq \alpha, \beta \leq 2$. E is the Young modulus, κ the Poisson ratio of the material and $\delta_{\alpha\beta}$ denotes the Kronecker symbol.

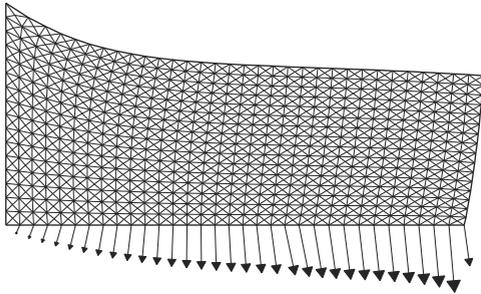


Fig. 4. Deformed mesh and contact interface forces on Γ_3 .

For computation we used the following data:

$$L_1 = 2 \text{ m}, \quad L_2 = 1 \text{ m}, \quad E = 1 \text{ N/m}^2, \quad \kappa = 0.3,$$

$$\mathbf{f}_0 = (0, 0) \text{ N/m}^2,$$

$$\mathbf{f}_2 = \begin{cases} (0, 0) \text{ N/m} & \text{on } \{2\} \times [0, 1], \\ (0, -0.3) \text{ N/m} & \text{on } [0, 2] \times \{1\}, \end{cases}$$

$$a = 0.004, \quad b = 0.002, \quad \alpha = 100, \quad S = 1 \text{ N},$$

stopping criterion: $\epsilon = 10^{-6}$.

We used a structural mesh of triangles generated by taking the equidistant lines parallel to both axes and taking two diagonals in each of the obtained rectangles. In Fig. 4 the deformed configuration as well as the contact interface forces on Γ_3 are plotted.

Now, for both numerical approaches presented in Sections 5 and 6, the tangential stresses and the tangential displacements on Γ_3 are plotted respectively in Fig. 5–8. In each of these figures, we plotted three curves corresponding to different values of the coefficients

a and b . The case $a = 0.004$ and $b = 0.002$ reflects the non-monotonicity of the friction law while the cases $a = b = 0.004$ and $a = b = 0.002$ correspond to the classical monotone Coulomb law of dry friction. According to Figs. 5–8, we can see that the results obtained by the two numerical approaches are very similar with respect to the tangential stresses and the tangential displacements on the contact boundary Γ_3 .

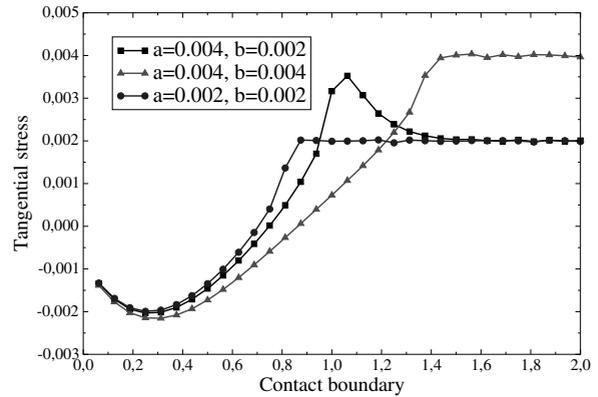


Fig. 5. Tangential stresses on Γ_3 for the PBM presented in Section 5.

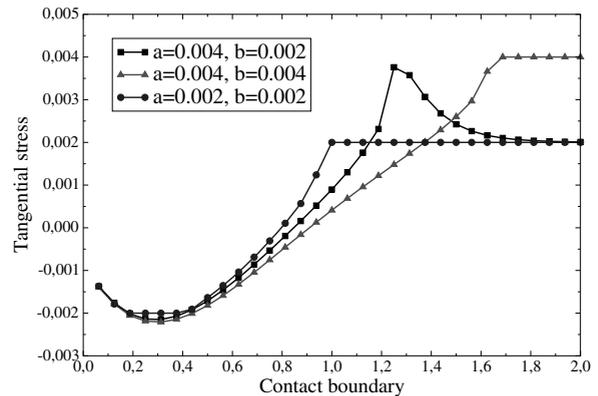


Fig. 6. Tangential stresses on Γ_3 for the algorithm presented in Section 6.

In Fig. 9, we plotted the tangential stresses with respect to tangential displacements at the node of coordinates $(2.0, 0.0)$ on Γ_3 during the iterations of the algorithm presented in Section 6. According to Fig. 9, we can find a nonmonotone behavior of the friction law characterized by the coefficients $a = 0.004$ and $b = 0.002$. For $a = b = 0.004$ and $a = b = 0.002$, we recover a monotone friction behavior.

The details of the computations are the following. The problem is discretized in 2048 elastic finite elements and 32 contact elements. The total number of degrees of freedom is equal to 2210. For the algorithm presented in Section 6, the simulation needs 17 “convex” iterations to solve the nonconvex problems. The total number of

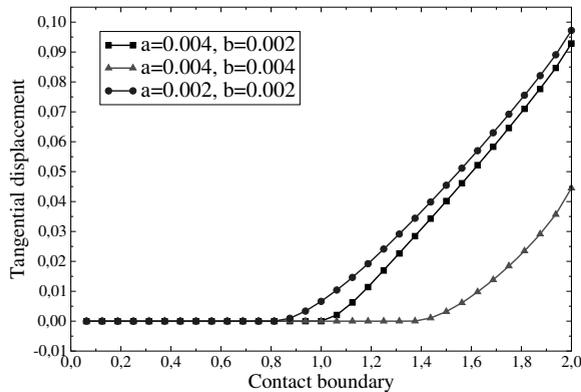


Fig. 7. Tangential displacements on Γ_3 for the PBM presented in Section 5.

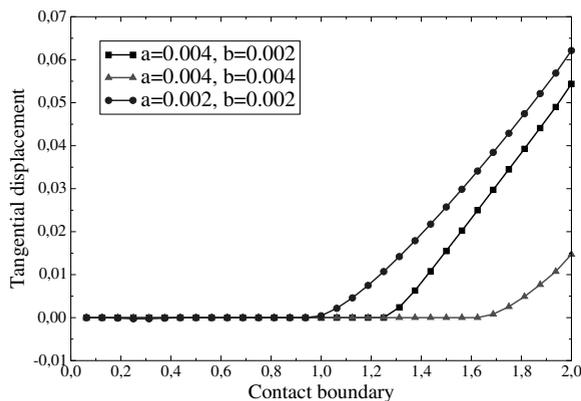


Fig. 8. Tangential displacements on Γ_3 for the algorithm presented in Section 6.

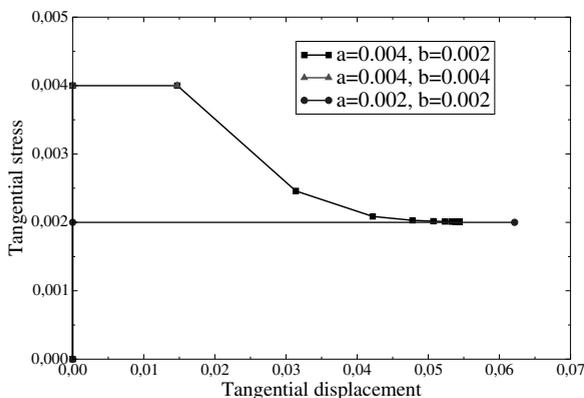


Fig. 9. Tangential stresses vs. tangential displacements at the node of coordinates $(2.0,0.0)$ on Γ_3 during the iterations of the algorithm presented in Section 6.

Newton iterations for the solution of the nonsmooth and nonconvex problem is equal to 42, whereas the number of iterations is equal to 8 for a nonsmooth convex problem characterized by $a = b = 0.004$ or $a = b = 0.002$. For the algorithm presented in Section 5, 541 iterations of the PBM loop were run for the case $a = b = 0.002$, 598

iterations for the case $a = b = 0.004$ and 433 iterations for the case $a = 0.004, b = 0.002$. The parameters of the PBM used in computations (see Mäkelä et al., 1999) are the following: $\gamma = 0.7, \bar{t} = 0.2, m_L = 0.2, m_R = 0.6, u_k = 0.01$. The trapezoidal formula for the numeric quadrature of the functional h defined by (45) was used, where each boundary edge was divided into 5000 equidistant intervals. For the bisection in the PBM, 400 steps were used.

We can note that the number of the PBM iterations is almost constant for both convex and nonconvex problems. This is in contrast to the algorithm presented in Section 6, where the number of iterations strongly depends on the convexity of the problem. We also underline that for convex subproblems solved by means of the augmented Lagrangian method the friction condition is considered a constraint, and in consequence it is fixed in each step of the method. In contrast, the PBM does not force the friction condition to be satisfied in each iteration step since the friction term is a part of the objective functional. We do not make comparisons in terms of CPU time because the algorithmic steps of the two methods are very different and for some of these steps the performance can be optimized. However, we note that the method presented in Section 5 is faster than the PBM since the conditions of friction are exactly satisfied in each step of iteration.

As prospects, we plan to refine the comparison of these methods by considering, for instance, less academic numerical examples and problems with other tangential and normal contact laws.

Acknowledgment

This work was conducted within the Polonium project on *Nonsmooth Analysis with Applications to Contact Mechanics* under the contract no. 7817/R09/R10 between Jagiellonian University and the University of Perpignan. This research was also supported by a Marie Curie International Research Staff Exchange Scheme Fellowship within the 7th European Community Framework Programme under the grant agreement no. PIRSES-GA-2011-295118. The last two authors were also supported in part by the National Science Center of Poland under the grant no. N N201 604640, by an international project co-financed by the Ministry of Science and Higher Education of the Republic of Poland under the grant no. W111/7.PR/2012, and by the National Science Center of Poland under the Maestro Advanced Project no. DEC-2012/06/A/ST1/00262.

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Received: 19 July 2012

Revised: 30 October 2012